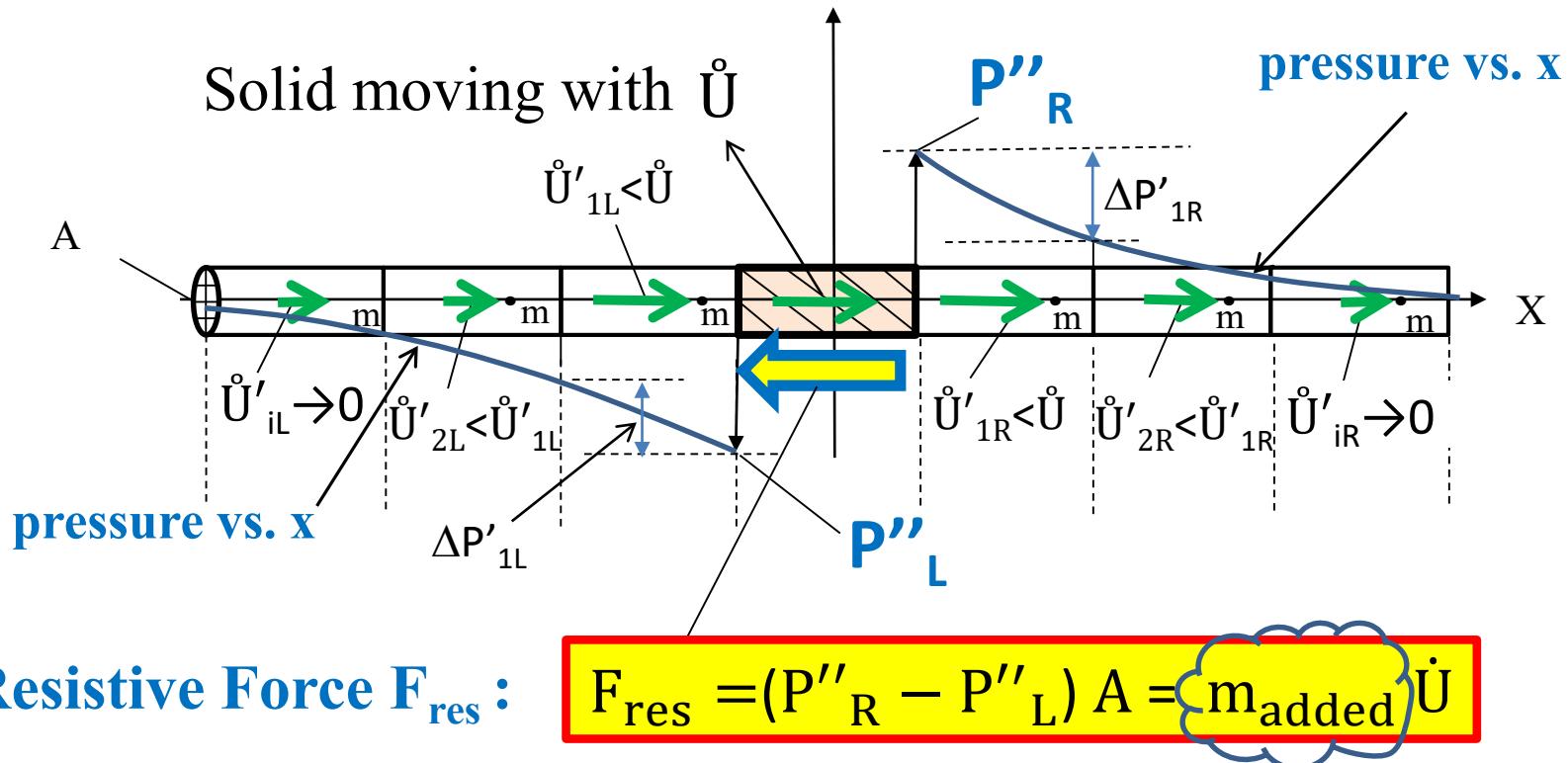


The concept of Added Mass (1/7)

Solid moving with acceleration \ddot{U} inside quiescent fluid

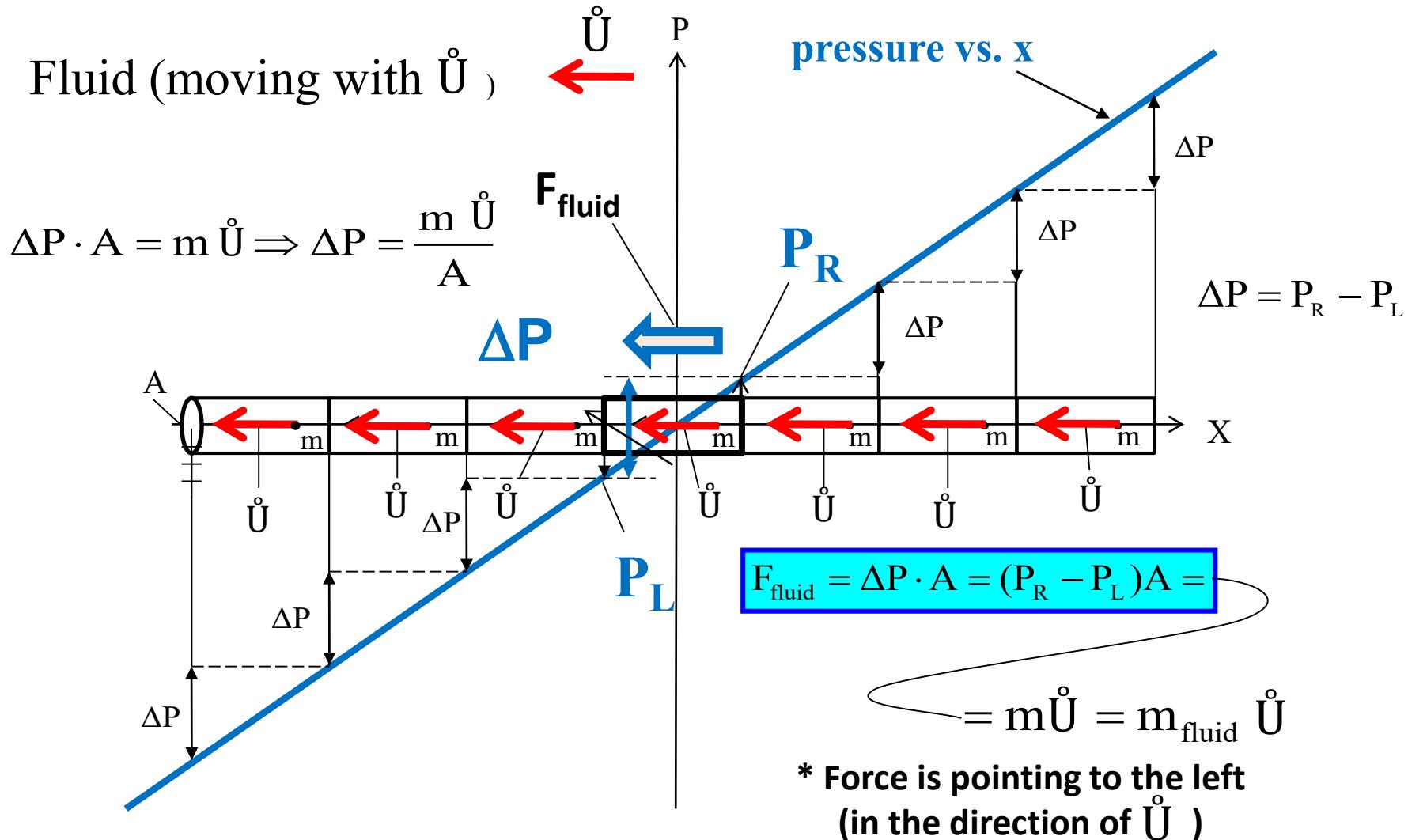


$$F_{res} = (P''_R - P''_L) A = \sum \Delta P'_{iR} A + \sum \Delta P'_{iL} A = \sum m \dot{U}'_{iR} + \sum m \dot{U}'_{iL}$$

The resistive force ($F_{res} = m_{\text{added}} \ddot{U}$) is needed in order to accelerate parts of the surrounding fluid which move with the body.

The concept of Added Mass (2/7)

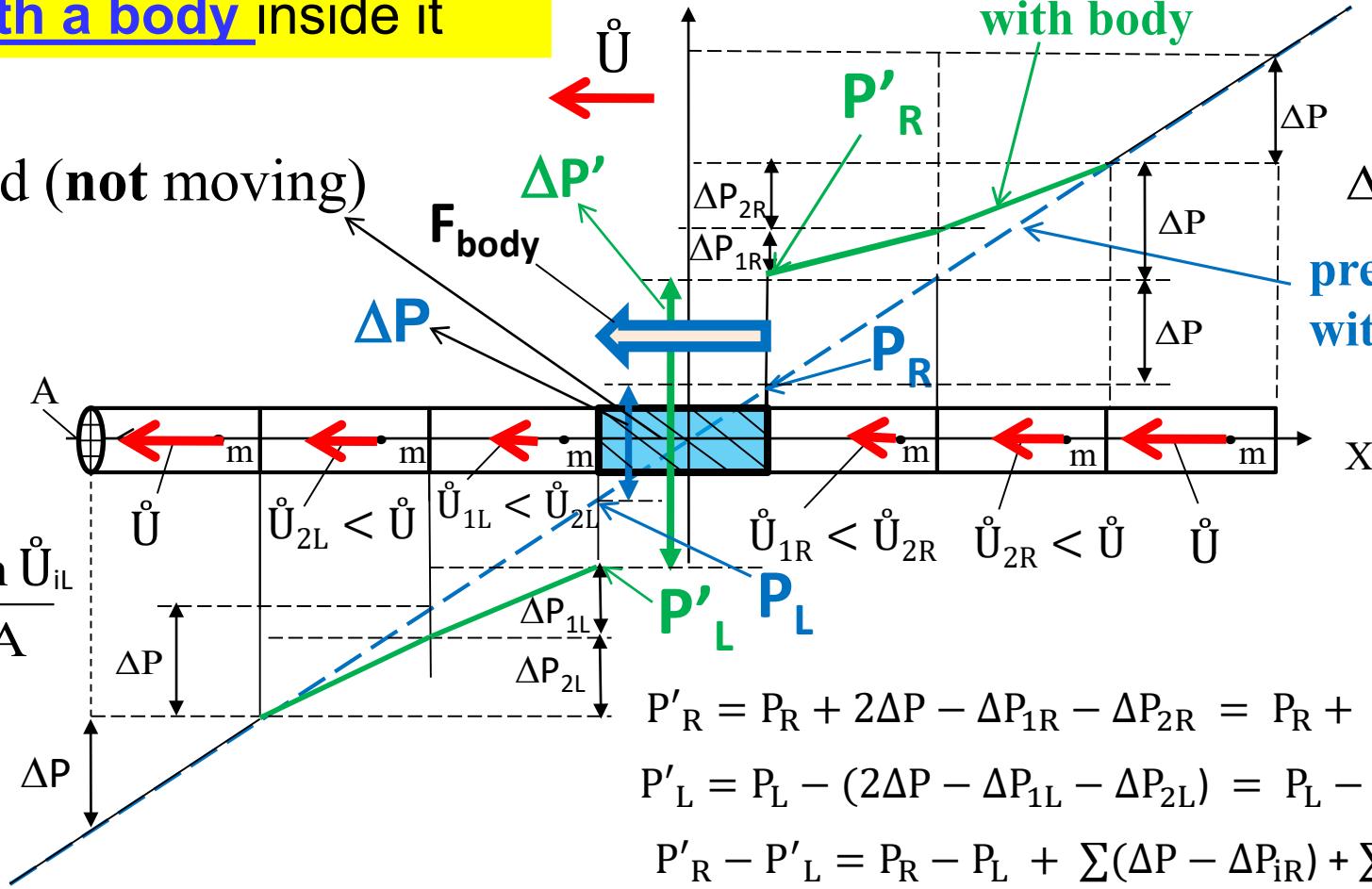
Fluid subject to acceleration without a body inside it



The concept of Added Mass (3/7)

Fluid subject to acceleration
with a body inside it

Solid (not moving)



pressure vs. x
with body

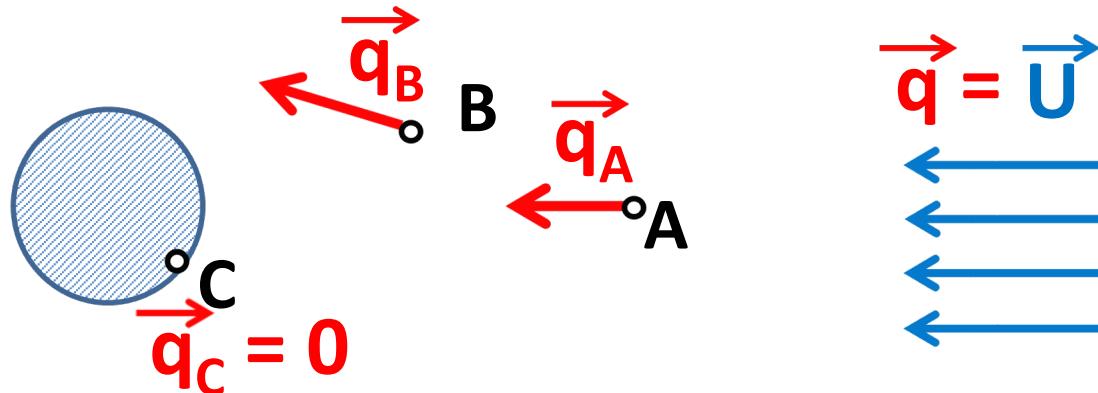
$$\Delta P_{iR} = \frac{m \dot{U}_{iR}}{A}$$

pressure vs. x
without body

The concept of Added Mass (4/7)

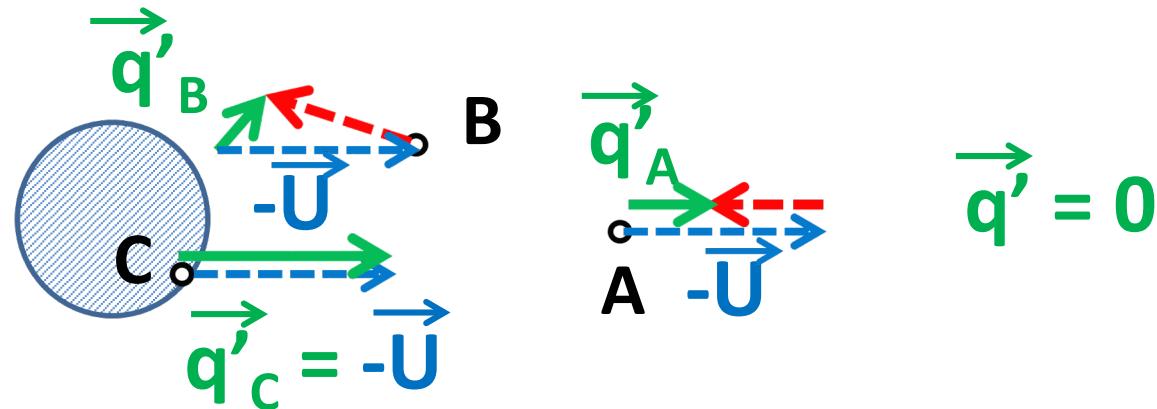
Two Different Frames of Reference (for the same problem!):

Body is stationary
and flow is moving
to the left with
speed U



$$\vec{q}' = \vec{q} - \vec{U}$$

Flow is stationary
(far upstream) and body
is moving to the right
with speed U



The concept of Added Mass (5/7)

$$F_{res} = (P''_R - P''_L) A = \sum m \dot{U}'_{iR} + \sum m \dot{U}'_{iL} = m_{added} \dot{U}$$

$$\dot{U}'_{iR} = -\dot{U}_{iR} - (-\dot{U}) = \dot{U} - \dot{U}_{iR}$$

$$\dot{U}'_{iL} = -\dot{U}_{iL} - (-\dot{U}) = \dot{U} - \dot{U}_{iL}$$

Thus:

$$\sum m (\dot{U} - \dot{U}_{iR}) + \sum m (\dot{U} - \dot{U}_{iL}) = m_{added} \dot{U}$$

$$F_{body} = m_{fluid} \dot{U} + \sum m (\dot{U} - \dot{U}_{iR}) + \sum m (\dot{U} - \dot{U}_{iL})$$

Thus:

$$F_{body} = m_{fluid} \dot{U} + m_{added} \dot{U} = (m_{fluid} + m_{added}) \dot{U}$$

$$F_{body} = F_{inertial} = C_M m_{fluid} \dot{U}$$

Inertia coefficient C_M

$$C_M = \frac{F_{body}}{F_{fluid}}$$

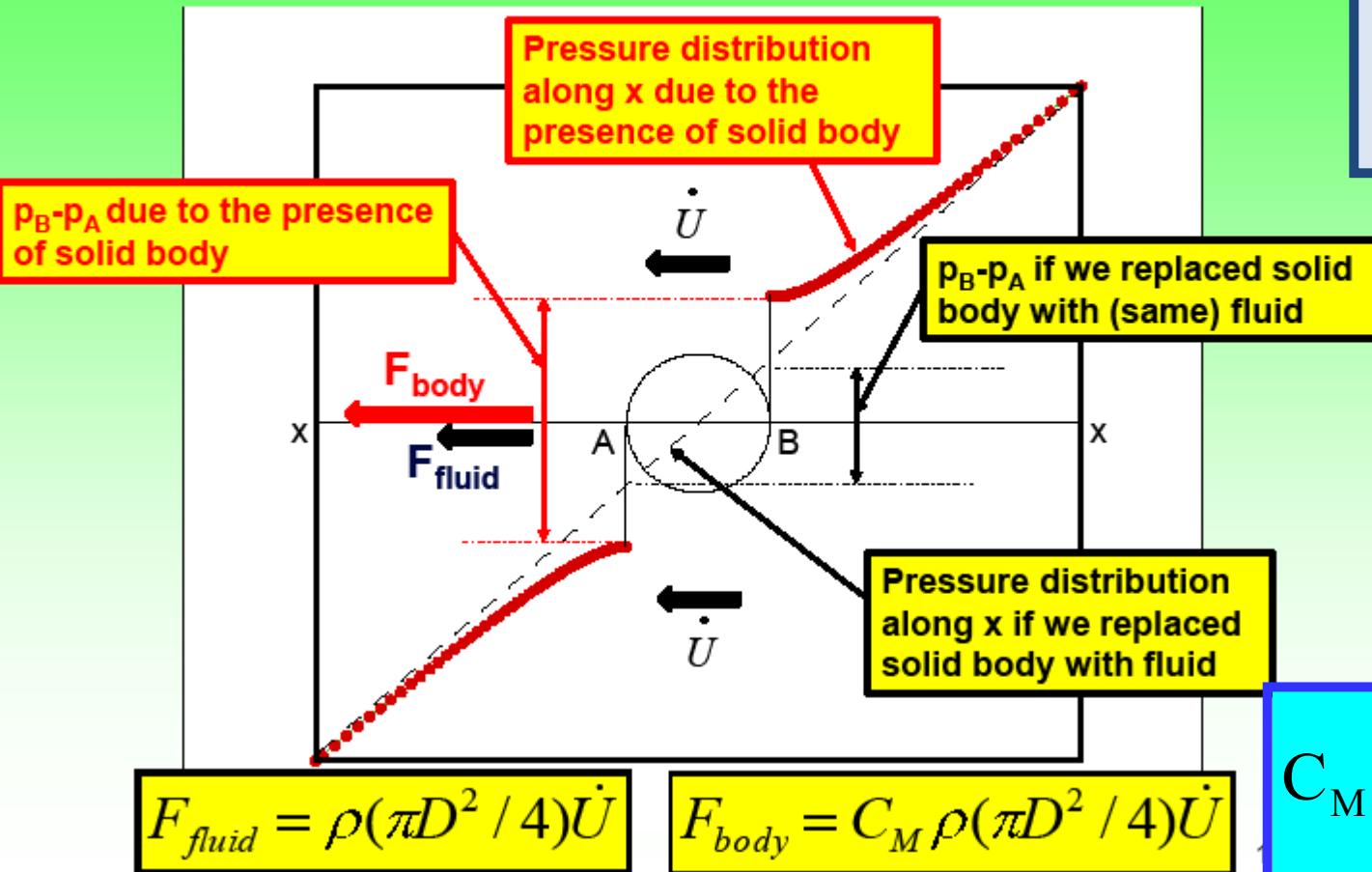
$$C_M = \frac{m_{fluid} + m_{added}}{m_{fluid}}$$

➤ m_{fluid} =mass of fluid displaced by body

➤ m_{added} =added mass; function of body shape and inflow direction

The concept of Added Mass (6/7)

Definition of inertia coefficient C_M



Cylinder subject to accelerated inflow.

Results from Inviscid flow simulation

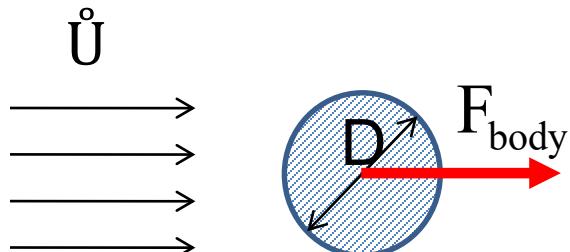
$$C_M = \frac{F_{body}}{F_{fluid}}$$

$$C_M = \frac{m_{fluid} + m_{added}}{m_{fluid}}$$

For inviscid flow around 2-D cylinder: $C_M=2$

The concept of Added Mass (7/7)

Summary:(all quantities are per unit width in 2-D)



$$F_{body} = C_M m_{fluid} \dot{U}$$

$m_{fluid} = \rho_{fluid} V_{fluid}$ = mass of displaced fluid

V_{fluid} = volume of displaced fluid

$$C_M = \text{inertia coefficient} = \frac{m_{fluid} + m_{added}}{m_{fluid}} = 1 + \frac{m_{added}}{m_{fluid}} = 1 + a$$

$$a = \frac{m_{added}}{m_{fluid}} = \text{added mass coefficient (depends on shape + direction of flow)}$$

For a cylinder (circle in 2-D) V_{fluid} =area of cross section = $\pi D^2/4$

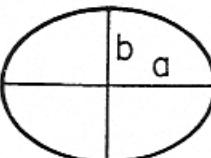
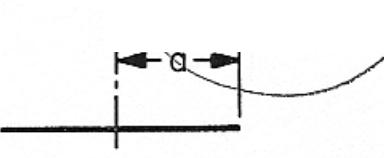
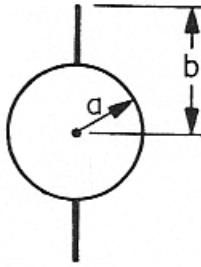
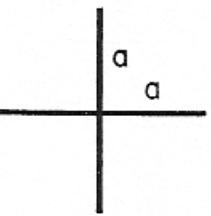
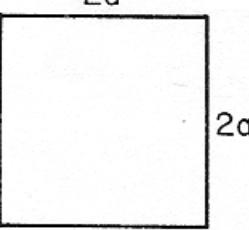
For a cylinder in inviscid unbounded flow: $C_M=2$ ($a=1$)

(Inviscid) Added Mass for other shapes:

m_{11} and m_{22} are the added masses when the flow is accelerated in the horizontal or the vertical axis, respectively. m_{66} is the added moment of inertia when the body rotates around an axis normal to the paper.

Table 4.3

Added-Mass Coefficients for Various Two-Dimensional Bodies.

| | | |
|--|--|--|
|  |  |  |
| $m_{11}: \pi \rho a^2$ | $\pi \rho b^2$ | 0 |
| $m_{22}: \pi \rho a^2$ | $\pi \rho a^2$ | $\pi \rho a^2$ |
| $m_{66}: 0$ | $\frac{1}{8} \pi \rho (a^2 - b^2)^2$ | $\frac{1}{8} \pi \rho a^4$ |
|  |  |  |
| $m_{11}: \pi \rho [a^2 + (b^2 - a^2)^2/b^2]$ | $\pi \rho a^2$ | 4.754 ρa^2 |
| $m_{22}: \pi \rho a^2$ | $\pi \rho a^2$ | 4.754 ρa^2 |
| $m_{66}: *$ | $\frac{2}{\pi} \rho a^4$ | 0.725 ρa^4 |

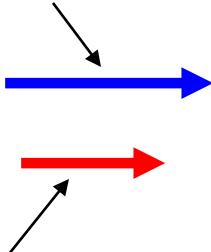
From Marine Hydrodynamics,
Newman, J.N., 1977

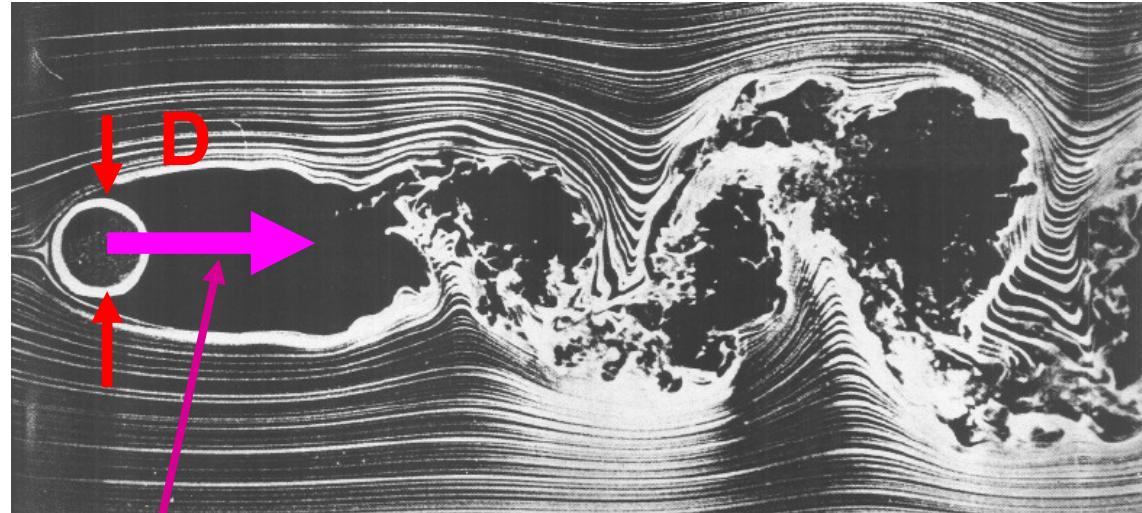
*For the finned circle the added moment of inertia is given by the formula

$$m_{66} = \rho a^4 (\pi^{-1} \csc^4 \alpha [2\alpha^2 - \alpha \sin 4\alpha + \frac{1}{2} \sin^2 2\alpha] - \pi/2)$$

Morison's equation for total force in the direction of wave propagation

[Morison, J. R.; O'Brien, M. P.; Johnson, J. W.; Schaaf, S. A. (1950), "The force exerted by surface waves on piles", Petroleum Transactions (American Institute of Mining Engineers) 189: 149–154]

Velocity, u

Acceleration, a



Total force = Viscous force + Inertial force

Total force
(per unit width)

Drag coefficient

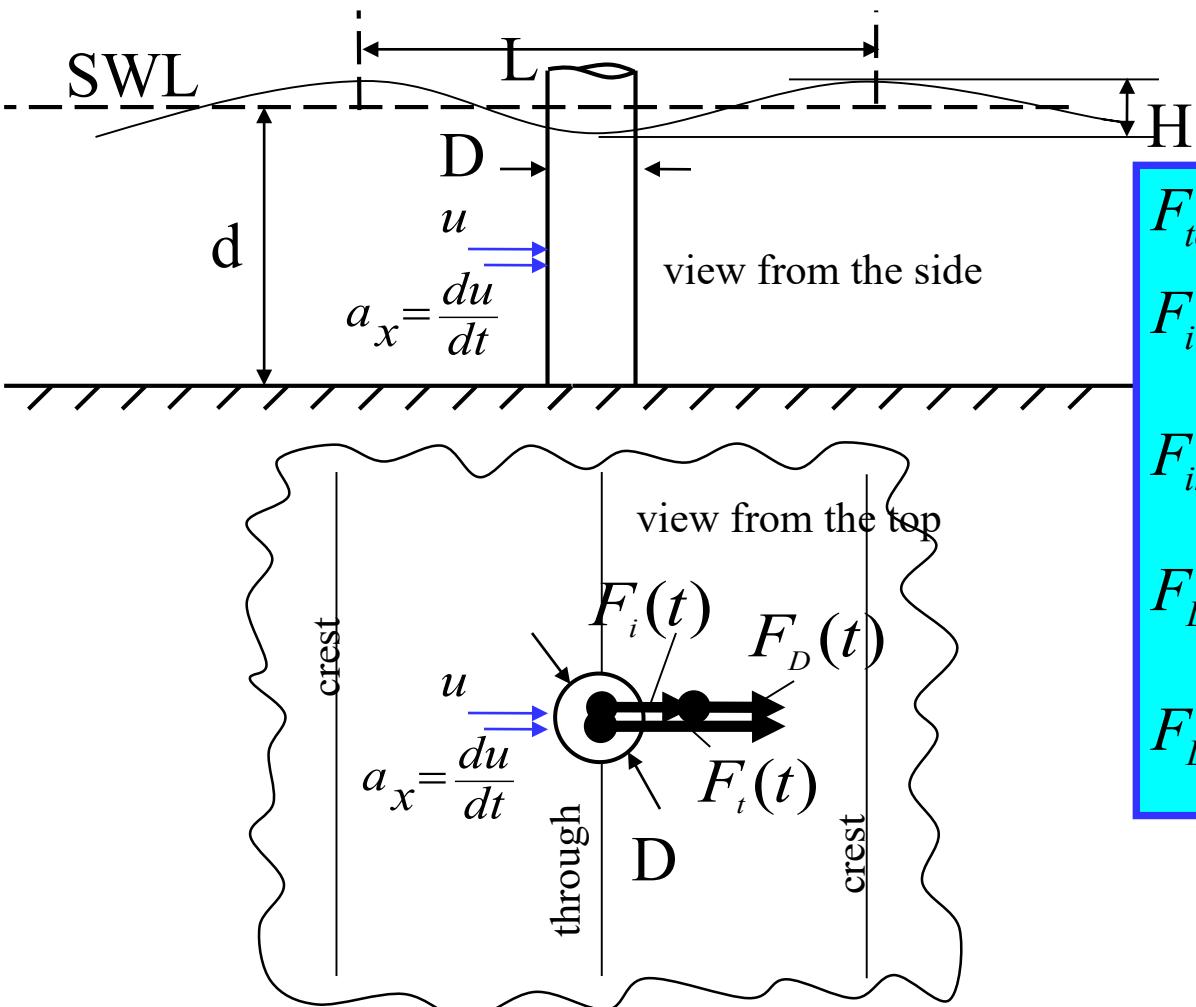
$$C_D \frac{1}{2} \rho D u |u|$$

$$C_M \rho \frac{\pi D^2}{4} a$$

Inertia coefficient

Application of Morison's equation to determine forces on vertical piles

Morison's equation is integrated over the length of the pile, after the values for u and a_x have been determined by either using linear or **non-linear wave theories**



$$\vartheta = -\omega t$$

$$F_{total}(t) = F_i(t) + F_D(t)$$

$$F_i(t) = F_{im} \cdot \sin(\vartheta)$$

$$F_{im} = C_M \cdot \rho g \cdot \frac{\pi D^2}{4} H \cdot K_{im}$$

$$F_D(t) = F_{Dm} \cdot |\cos \vartheta| \cos \vartheta$$

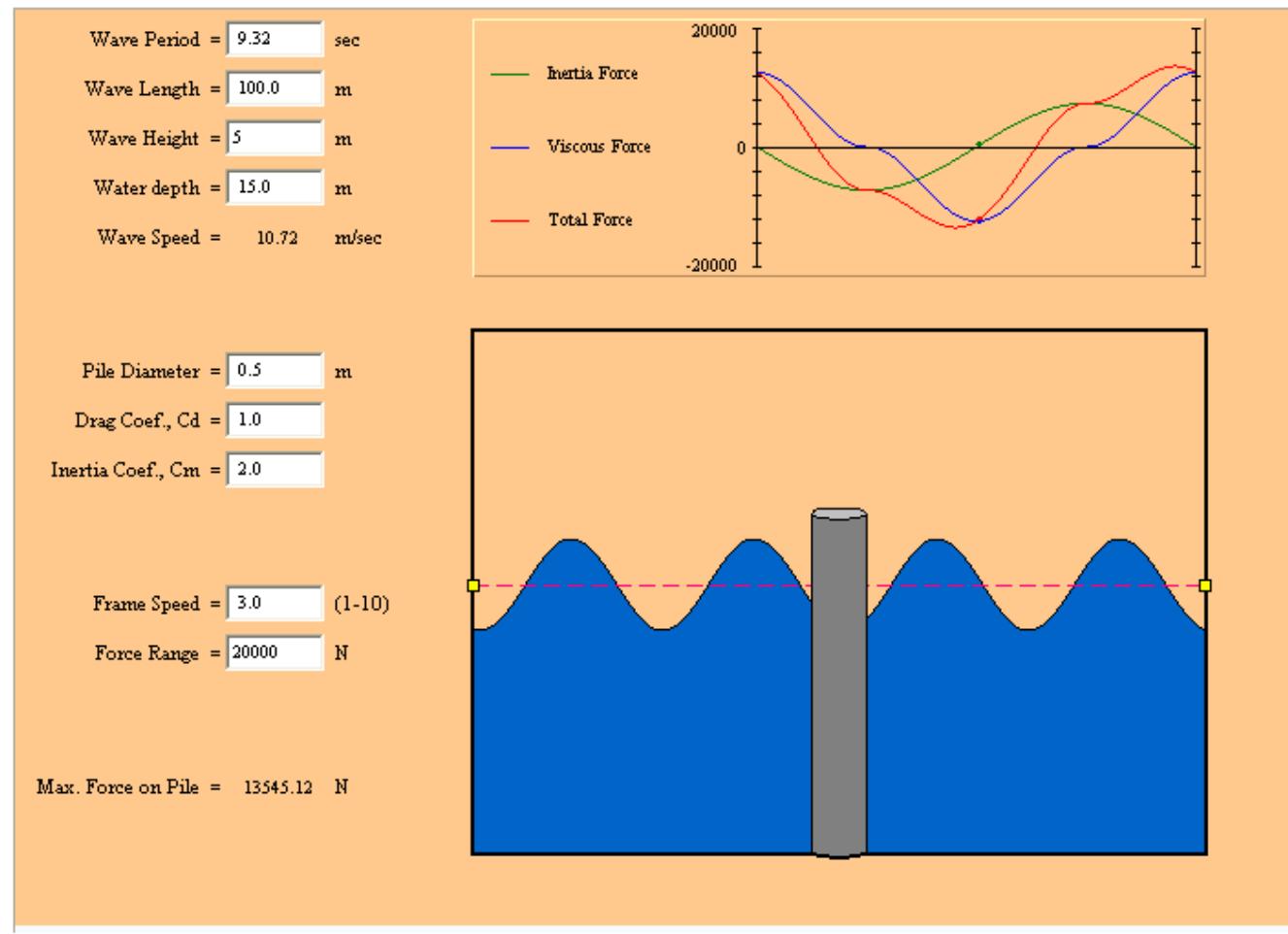
$$F_{Dm} = C_D \frac{1}{2} \rho g D H^2 \cdot K_{Dm}$$

$$K_{im} = \frac{1}{2} \tanh\left(\frac{2\pi d}{L}\right)$$

$$K_{Dm} = \frac{1}{8} \left(1 + \frac{4\pi d / L}{\sinh[4\pi d / L]} \right)$$

The wave forces applet sums-up the forces per pile slice over the pile length

http://cavity.ce.utexas.edu/kinnas/wow/public_html/waveroom/Applet/WaveForces/WaveForces.html



Typical values of the drag and inertia coefficients

From API's (American Petroleum Institute)

Recommended Practice 2A-WSD (Dec. 2000)

- $C_D = 0.65$ and $C_M=1.6$ for smooth piles
- $C_D = 1.05$ and $C_M=1.2$ for rough piles (due to marine growth)

Note: The diameter of the pile, D, also increases with marine growth



Total Force on Pile

(in the direction of wave propagation)

Total force = Viscous force + Inertial force

Morison's equation

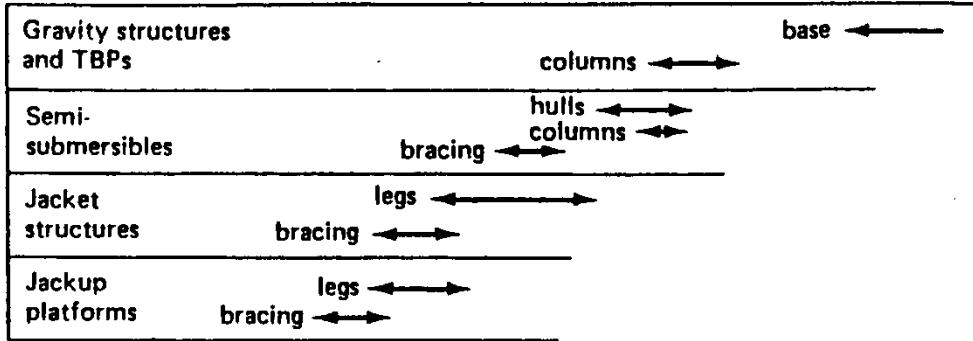
$$\rightarrow \sim C_D \rho D H^2 + \sim C_M \rho D^2 H$$

Drag coefficient

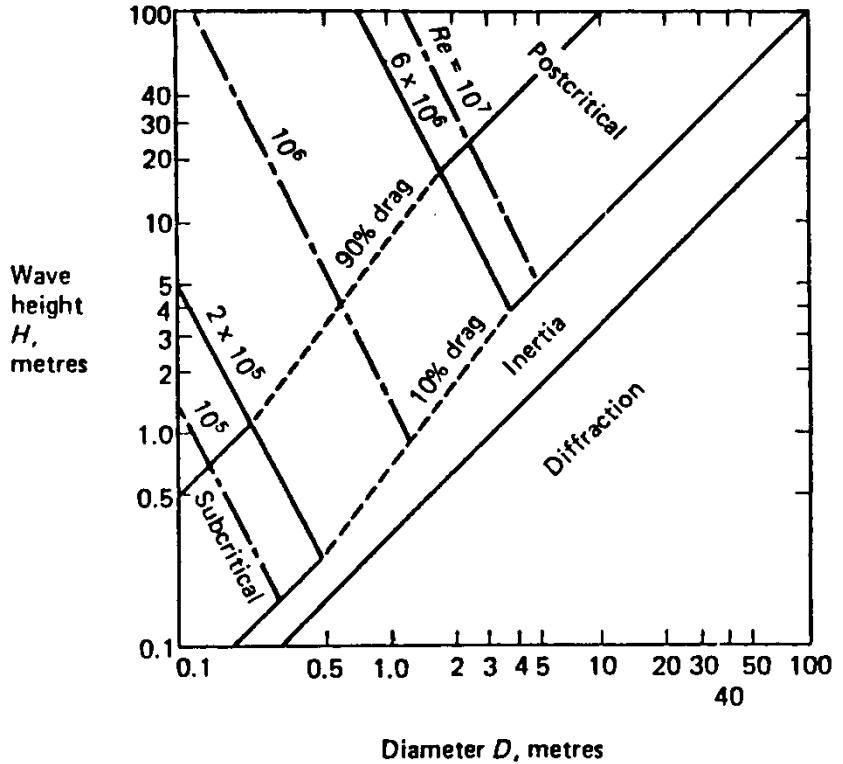
Inertia coefficient

$$\frac{\text{Viscous force}}{\text{Inertial force}} \sim \frac{C_D}{C_M} \frac{H}{D}$$

As $H \uparrow$ or $D \downarrow$ or $H/D \uparrow$ the viscous forces become more important



Effect of wave height H and diameter of element D on importance of viscous forces

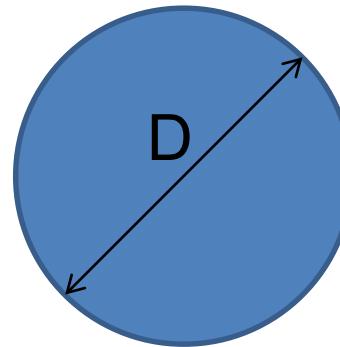
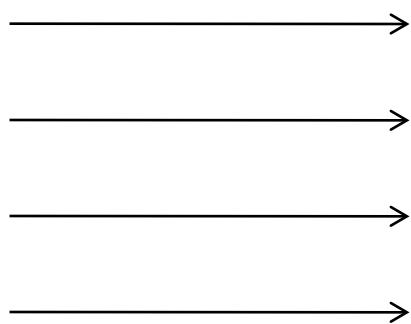


Loading regimes at still water level (from Hogben⁴)

An assessment of Morison's equation using CFD (Computational Fluid Dynamics)

Two Dimensional Cylinder in Oscillatory Flow

$$U = U_m \cdot \cos(\omega t)$$



Two important numbers:

- **Re (Reynolds No)= $U_m D / \nu$**
- **KC (Keulegan-Carpenter No)= $U_m T / D$ ($T=2\pi/\omega$)**
~(distance the particles travel in T)/D

Morison's Equation

The inline force (force in the direction of the flow) is the sum of the drag force and the inertia force (per unit width)

$$F = \frac{1}{2} \rho C_D D |U| U + \frac{1}{4} \rho \pi D^2 C_M \frac{dU}{dt}$$

$$U = U_m \cdot \cos(\omega t)$$

C_M is the inertia coefficient

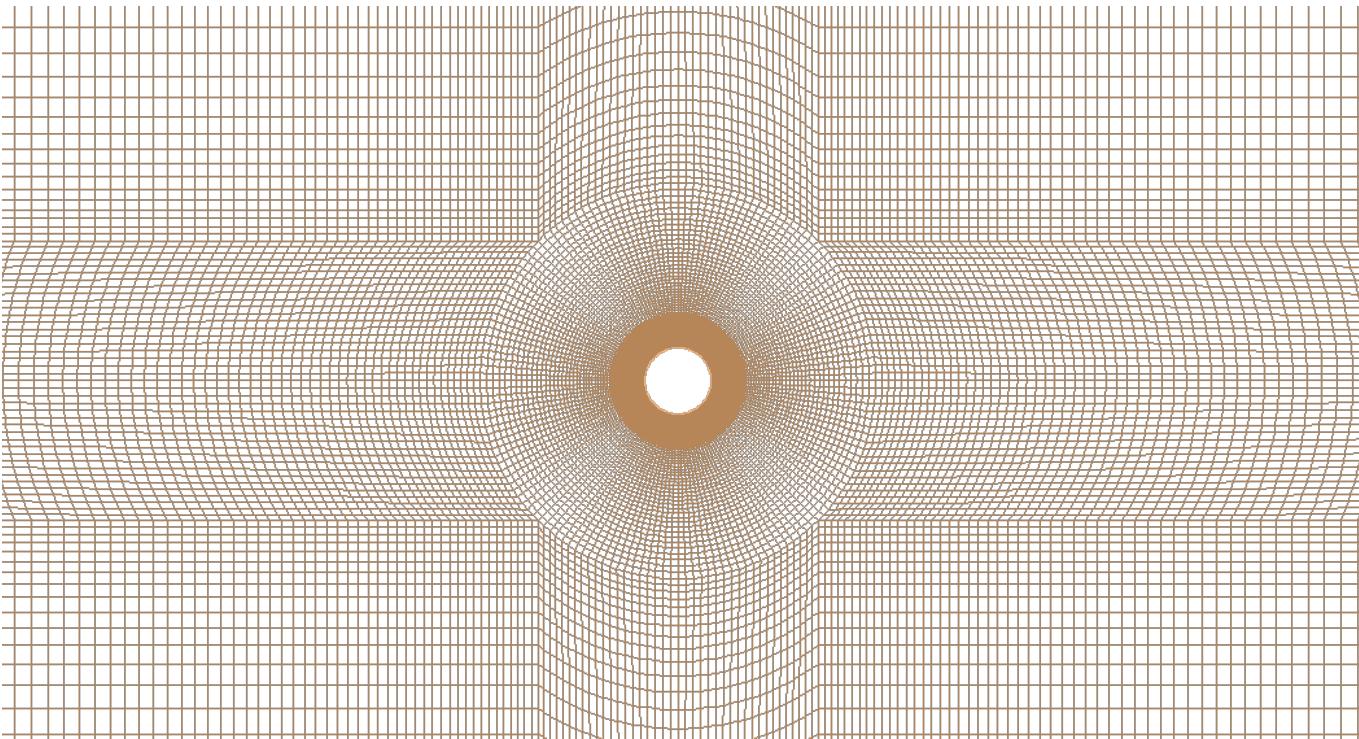
C_D is the drag coefficient

- we also define:

$$C_x = \frac{F}{\frac{\rho}{2} U_m^2 D}$$

Grid in Fluent

Structured mesh is used in the calculation domain



Mesh Info:

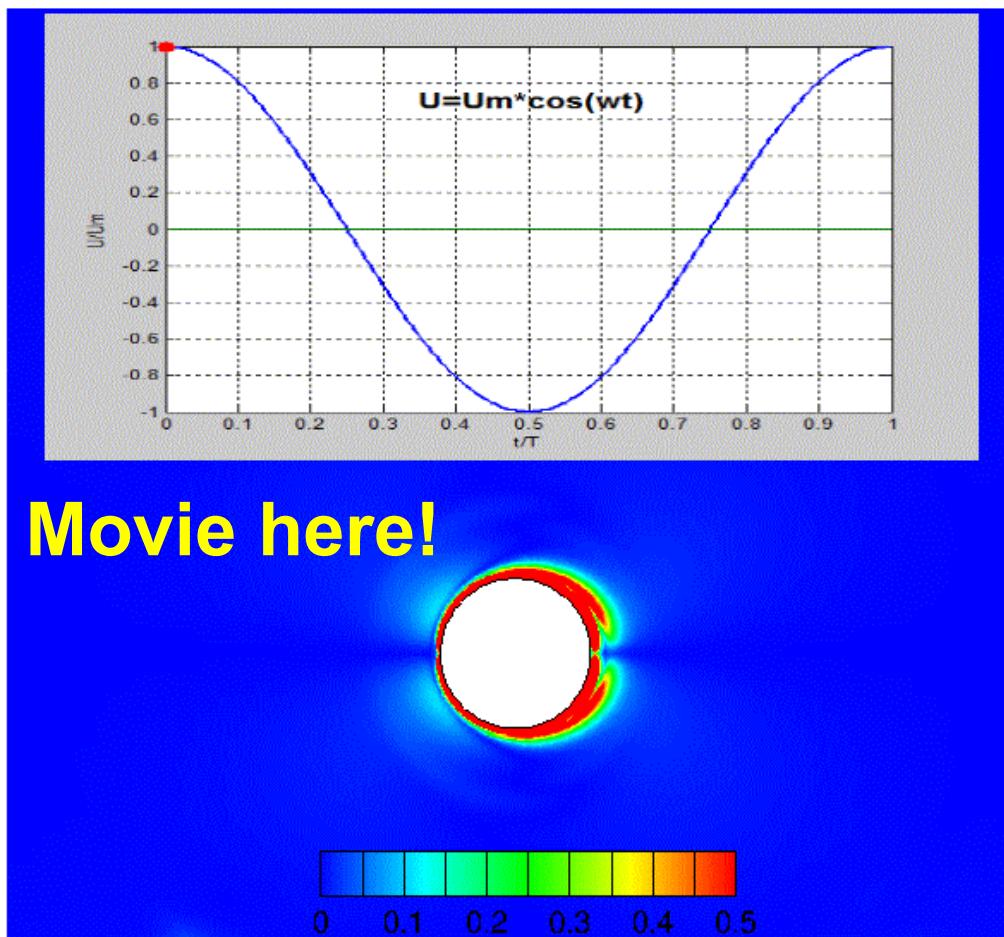
Cells: 76680

Faces: 154190

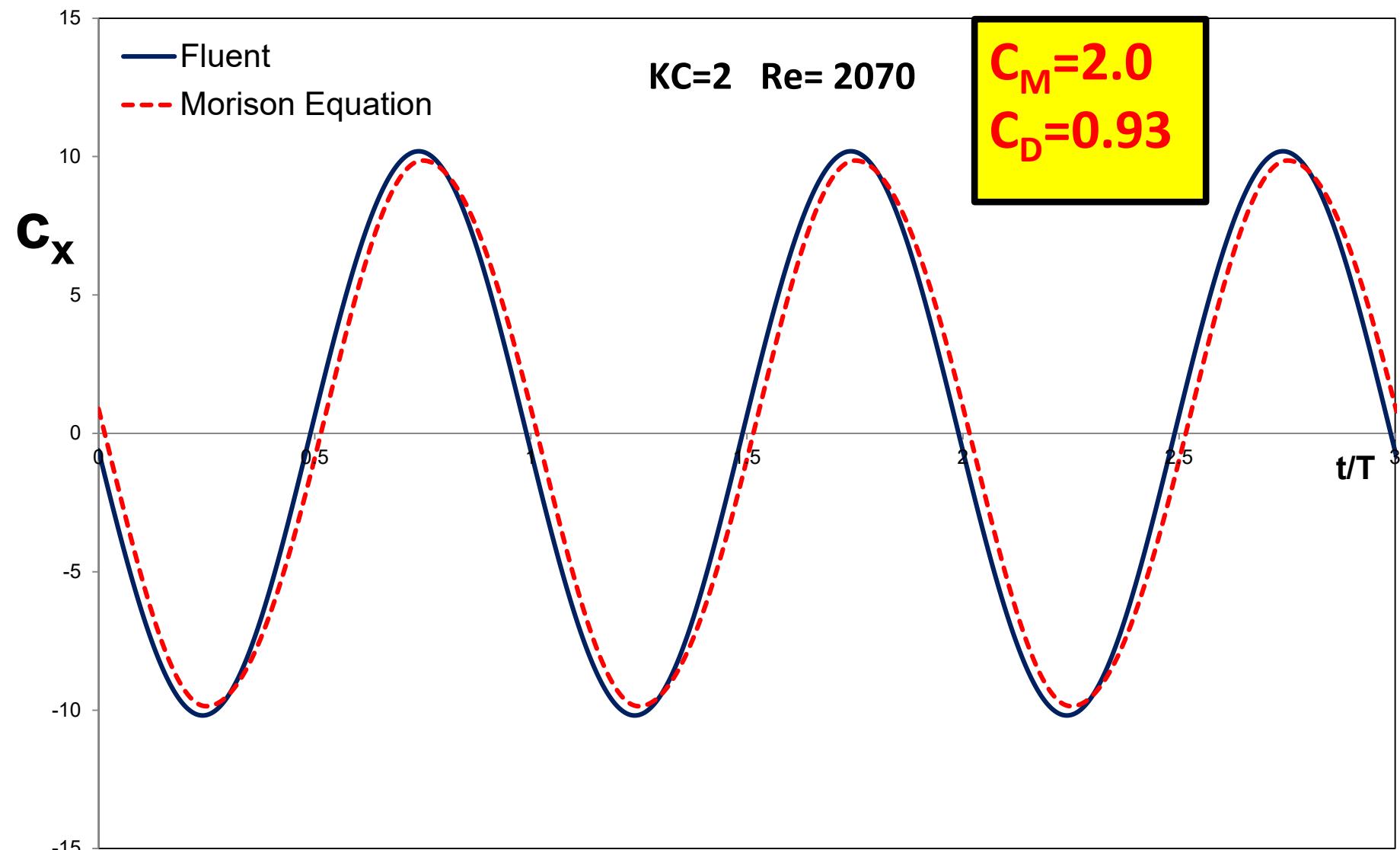
Nodes: 77510

Predicted flow (vorticity) by Fluent: KC=2, Re=1070

(click on the movie to play)

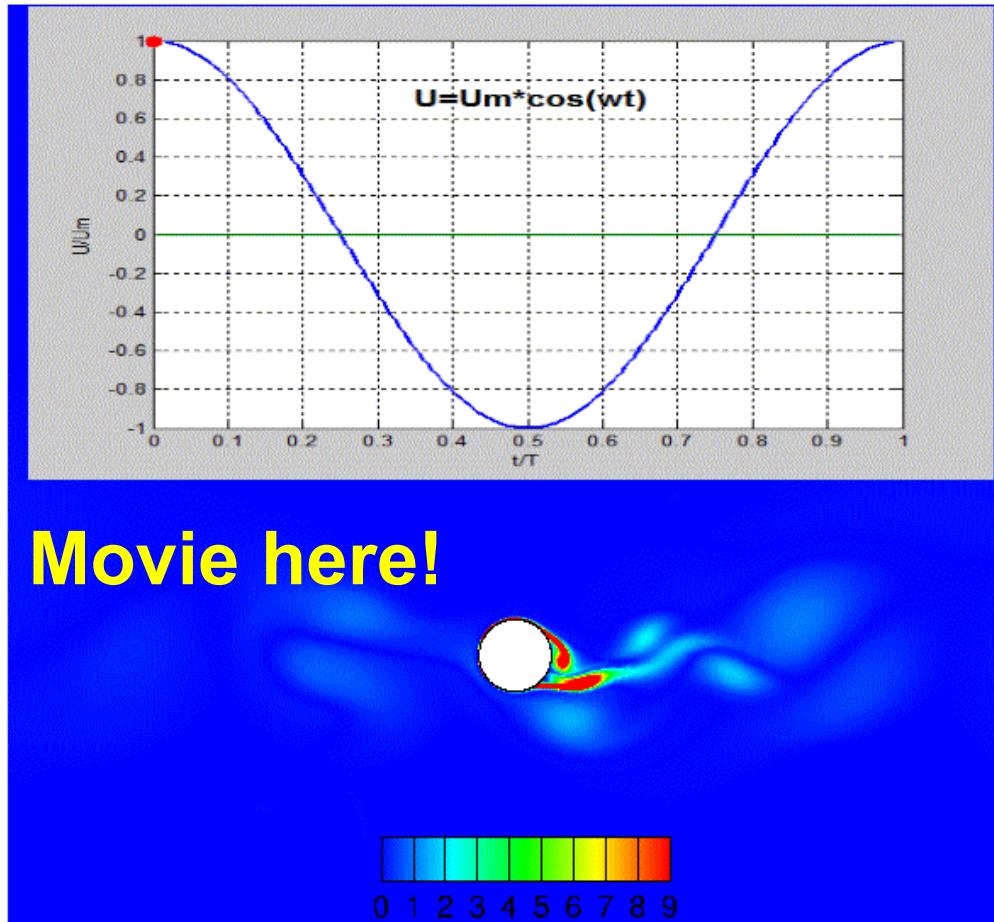


Case I: KC=2 Re=1070

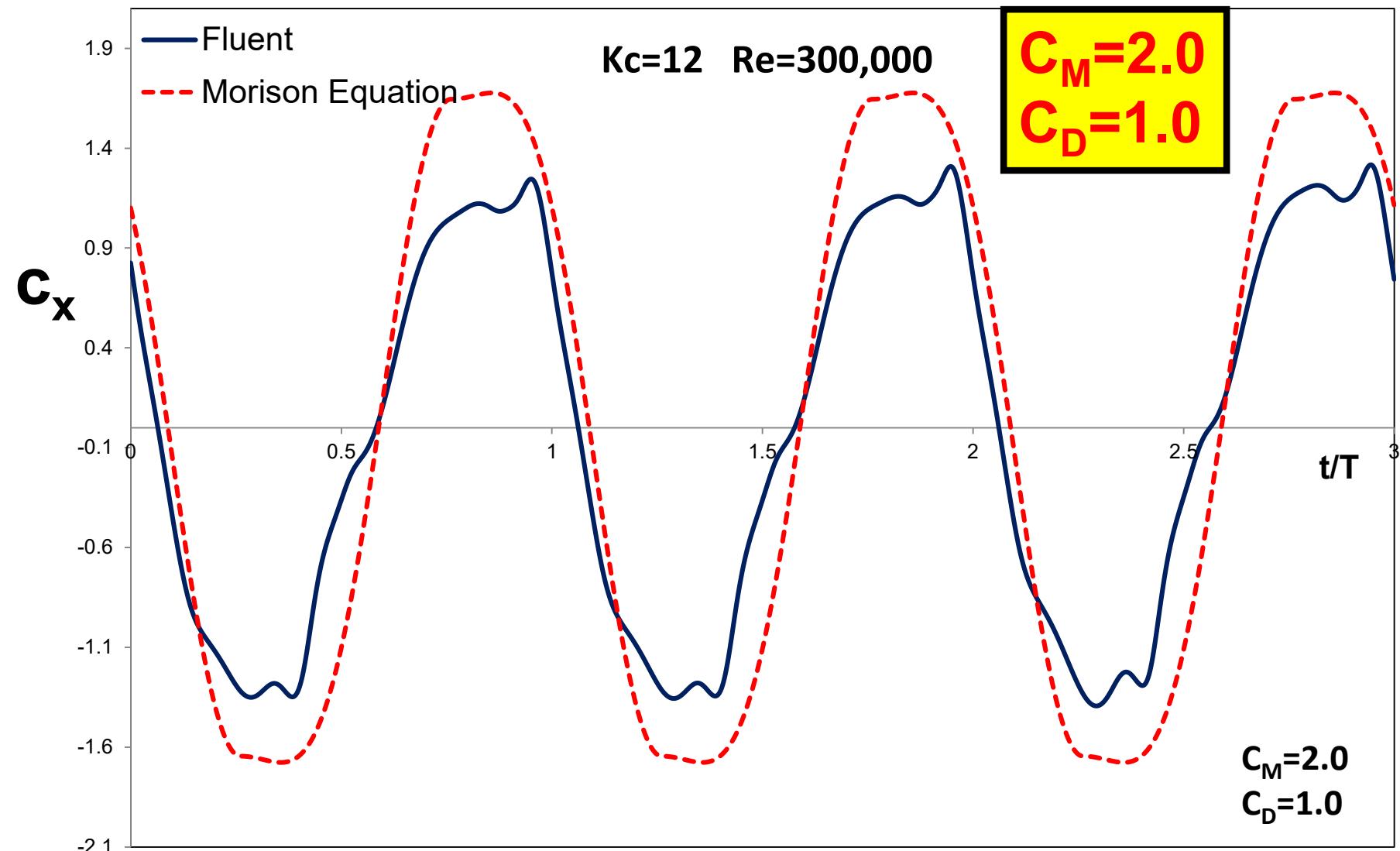


Predicted flow (vorticity) by Fluent: KC=12 Re=300,000

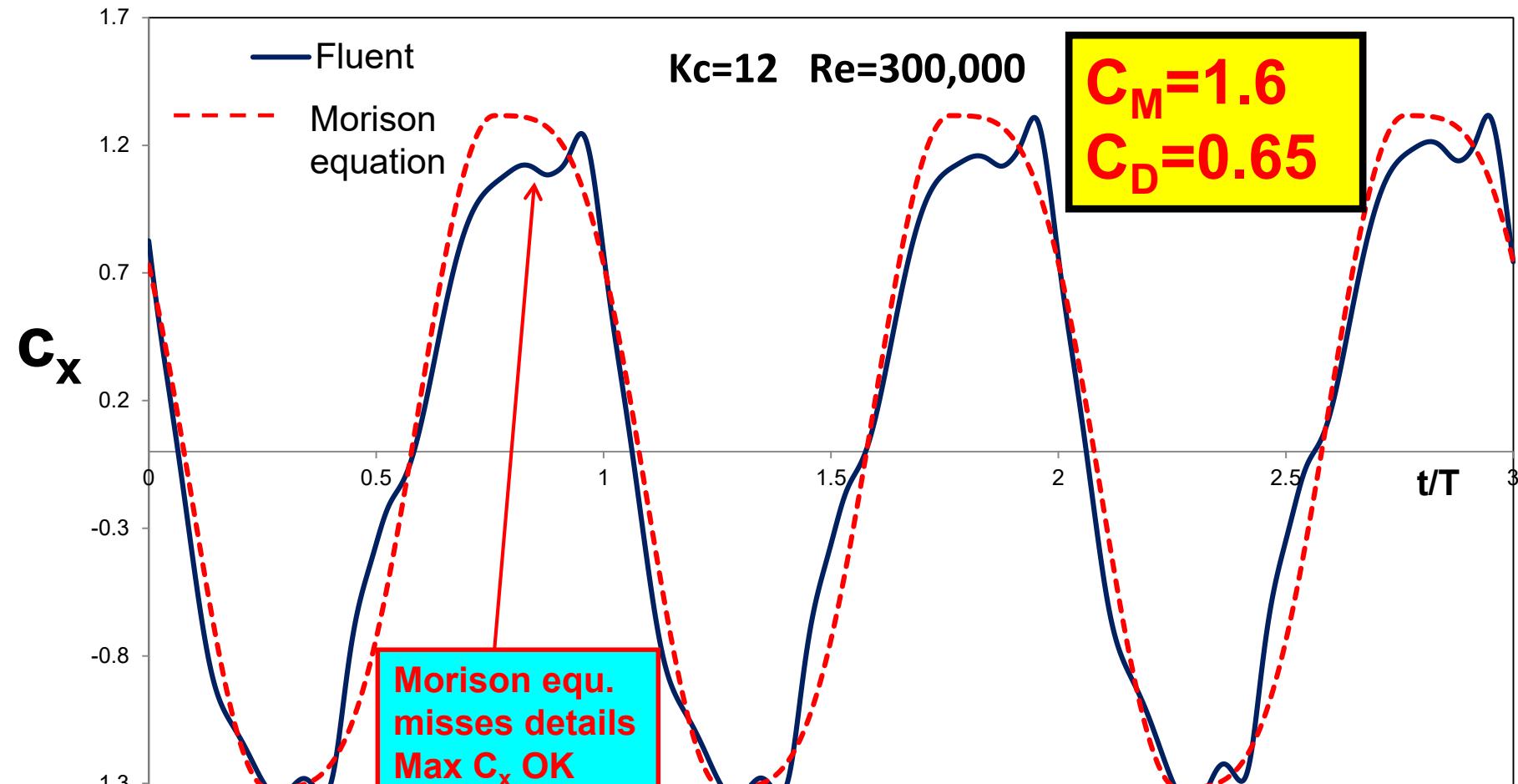
(click on the movie to play)



Case II: KC=12 Re=300,000



Case II: KC=12 Re=300,000



Maybe due to luck...more needs to be done!!! Some newer simulations will be shown in the lecture on Computational Hydrodynamics