



CE 319 F

Daene McKinney

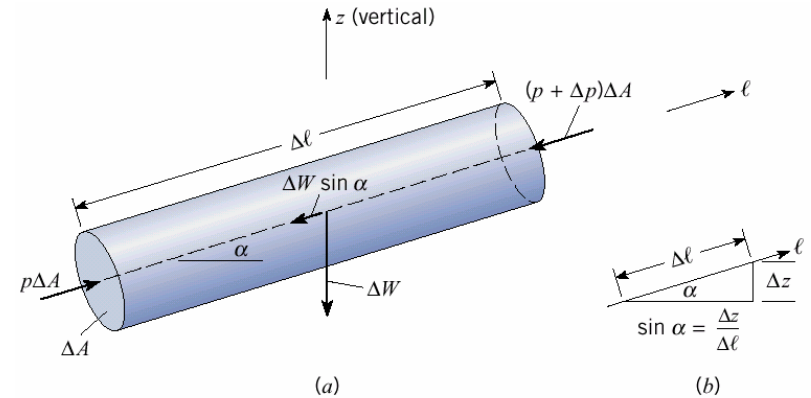
Elementary Mechanics of Fluids

Bernoulli
Equation



Euler Equation

- Fluid element accelerating in l direction & acted on by pressure and weight forces only (no friction)
- Newton's 2nd Law



$$\sum F_l = Ma_l$$

$$p\Delta A - (p + \Delta p)\Delta A - \Delta W \sin \alpha = \rho \Delta \ell \Delta A a_l$$

$$p - (p + \Delta p) - \gamma \Delta \ell \sin \alpha = \rho \Delta \ell a_l$$

$$-\frac{dp}{dl} - \rho g \frac{dz}{dl} = \rho a_l$$

$$-\frac{d}{dl} \left(\frac{p}{\gamma} + z \right) = \frac{a_l}{g}$$

Ex (5.1)

- **Given:** Steady flow. Liquid is decelerating at a rate of $0.3g$.
- **Find:** Pressure gradient in flow direction in terms of specific weight.

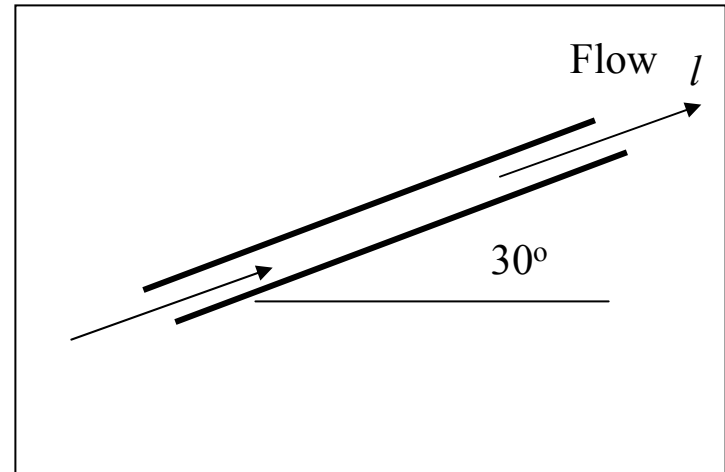
$$-\frac{d}{dl}\left(\frac{p}{\gamma} + z\right) = \frac{a_l}{g}$$

$$\frac{dp}{dl} = -\frac{\gamma}{g}a_l - \gamma \frac{dz}{dl}$$

$$= -\frac{\gamma}{g}(-0.3g) - \gamma \sin 30^\circ$$

$$= \gamma(0.3 - 0.5)$$

$$\frac{dp}{dl} = -0.2\gamma$$



EX (5.3)

- **Given:** $\gamma = 10 \text{ kN/m}^3$, $p_B - p_A = 12 \text{ kPa}$.
- **Find:** Direction of fluid acceleration.

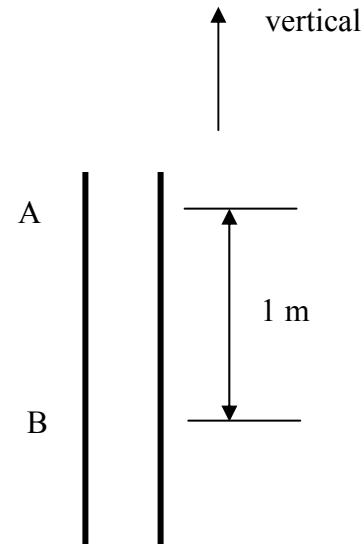
$$-\frac{d}{dz}\left(\frac{p}{\gamma} + z\right) = \frac{a_z}{g}$$

$$a_z = -g\left(\frac{1}{\gamma} \frac{dp}{dz} + \frac{dz}{dz}\right)$$

$$a_z = -g\left(\frac{p_A - p_B}{\gamma} + 1\right)$$

$$a_z = -g\left(\frac{-12,000}{10,000} + 1\right)$$

$$a_z = g(1.2 - 1) > 0 \quad (\text{acceleration is up})$$



HW (5.7)

- Ex (5.6) What pressure is needed to accelerate water in a horizontal pipe at a rate of 6 m/s²?

$$-\frac{d}{dl}\left(\frac{p}{\gamma} + z\right) = \frac{a_l}{g}$$

$$-\frac{dp}{dl} - \gamma \frac{dz}{dl} = \rho a_l$$

$$-\frac{dp}{dl} = \rho a_l = 1000 \text{ kg} / \text{m}^3 * 6 \text{ m} / \text{s}^2$$

$$\frac{dp}{dl} = -6000 \text{ N} / \text{m}^3$$

Ex (5.10)

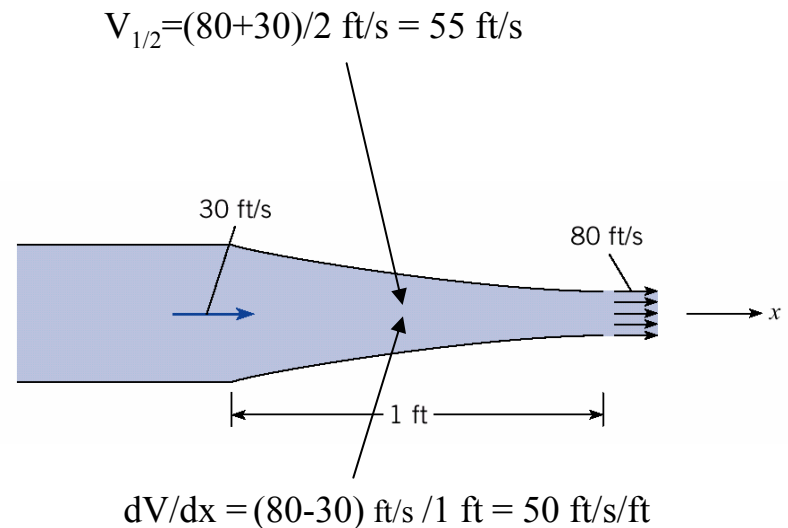
- **Given:** Steady flow. Velocity varies linearly with distance through the nozzle.
- **Find:** Pressure gradient $\frac{1}{2}$ -way through the nozzle

$$-\frac{d}{dx}\left(\frac{p}{\gamma} + z\right) = \frac{a_x}{g}$$

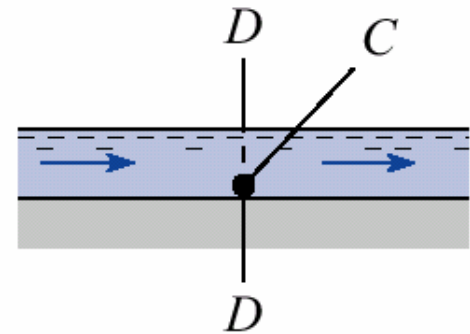
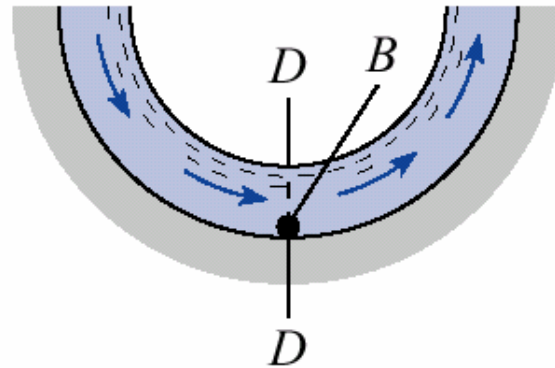
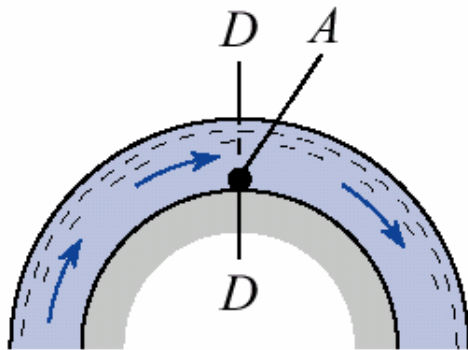
$$\frac{dp}{dx} = -\rho a_x = -\rho\left(V \frac{dV}{dx}\right)$$

$$= (-1.94 \text{ slugs} / \text{ft}^3) * (55 \text{ ft} / \text{s}) * (50 \text{ ft} / \text{s} / \text{ft})$$

$$= -5,355 \text{ lbf} / \text{ft}^2 / \text{ft}$$



HW (5.11)



Bernoulli Equation

$$\begin{aligned}-\frac{d}{ds}\left(\frac{p}{\gamma} + z\right) &= \frac{1}{g}a_t \\ &= \frac{1}{g}V\frac{dV}{ds} \\ &= \frac{d}{ds}\left(\frac{V^2}{2g}\right)\end{aligned}$$

$$\frac{d}{ds}\left(\frac{p}{\gamma} + z + \frac{V^2}{2g}\right) = 0$$

$$\frac{p}{\gamma} + z + \frac{V^2}{2g} = \text{Constant}$$

- Consider steady flow along streamline
- s is along streamline, and t is tangent to streamline

$$\frac{p}{\gamma} + z = \text{Piezometric head}$$

$$\frac{V^2}{2g} = \text{Velocity (dynamic) head}$$

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

Ex (5.47)

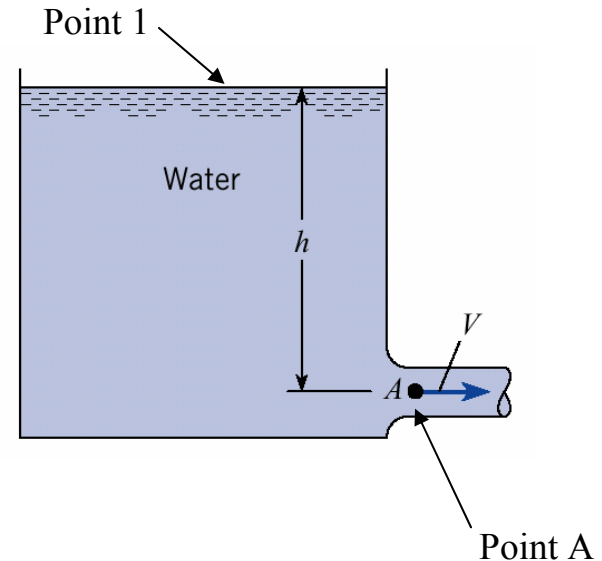
- **Given:** Velocity in outlet pipe from reservoir is 6 m/s and $h = 15$ m.
- **Find:** Pressure at A.
- **Solution:** Bernoulli equation

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_A}{\gamma} + z_A + \frac{V_A^2}{2g}$$

$$\frac{0}{\gamma} + h + \frac{0}{2g} = \frac{p_A}{\gamma} + 0 + \frac{V_A^2}{2g}$$

$$p_A = \gamma \left(h - \frac{V_A^2}{2g} \right) = 9810 \left(15 - \frac{18}{9.81} \right)$$

$$p_A = 129.2 \text{ kPa}$$



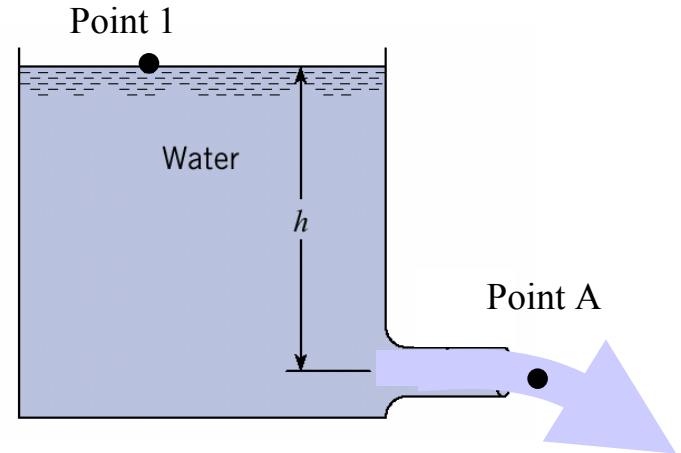
Example

- **Given:** $D=30$ in, $d=1$ in, $h=4$ ft
- **Find:** V_A
- **Solution:** Bernoulli equation

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_A}{\gamma} + z_A + \frac{V_A^2}{2g}$$

$$\frac{0}{\gamma} + h + \frac{0}{2g} = \frac{0}{\gamma} + 0 + \frac{V_A^2}{2g}$$

$$\begin{aligned} V_A &= \sqrt{2gh} \\ &= 16 \text{ ft/s} \end{aligned}$$



Example – Venturi Tube

- **Given:** Water 20°C, $V_1=2$ m/s, $p_1=50$ kPa, $D=6$ cm, $d=3$ cm
- **Find:** p_2 and p_3
- **Solution:** **Continuity Eq.**

$$V_1 A_1 = V_2 A_2$$

$$V_2 = V_1 \frac{A_1}{A_2} = V_1 \left(\frac{D}{d} \right)^2$$

- **Bernoulli Eq.**

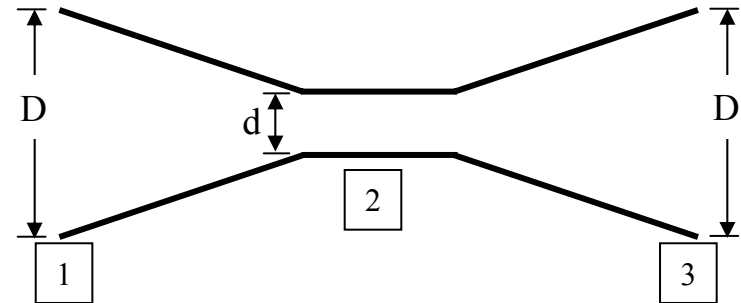
$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

$$p_2 = p_1 + \frac{\rho}{2}(V_1^2 - V_2^2)$$

$$= p_1 + \frac{\rho}{2}[1 - (D/d)^4]V_1^2$$

$$= 150,000 + \frac{1000}{2}[1 - (6/3)^4]2^2 \text{ Pa}$$

$$p_2 = 120 \text{ kPa}$$



Nozzle: velocity increases, pressure decreases

Diffuser: velocity decreases, pressure increases

Similarly for $2 \rightarrow 3$, or $1 \rightarrow 3$

$$p_3 = 150 \text{ kPa}$$

Pressure drop is fully recovered, since we assumed no frictional losses

Knowing the pressure drop $1 \rightarrow 2$ and d/D , we can calculate the velocity and flow rate

$$V_2 = \sqrt{\frac{2(p_1 - p_2)}{\rho[1 - (d/D)^4]}}$$

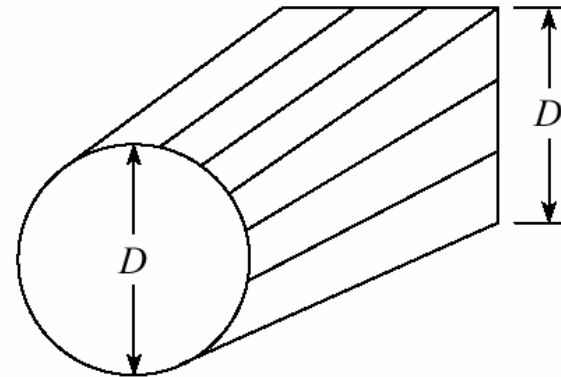
Ex (5.48)

- **Given:** Velocity in circular duct = 100 ft/s, air density = 0.075 lbm/ft³.
- **Find:** Pressure change between circular and square section.
- **Solution:** Continuity equation

$$V_c A_c = V_s A_s$$

$$100\left(\frac{\pi}{4} D^2\right) = V_s D^2$$

$$V_s = 100\left(\frac{\pi}{4}\right) = 78.54 \text{ ft/s}$$



Air conditioning ($\sim 60^\circ\text{F}$)

- Bernoulli equation

$$\frac{p_c}{\gamma} + z_c + \frac{V_c^2}{2g} = \frac{p_s}{\gamma} + z_s + \frac{V_s^2}{2g}$$

$$p_c - p_s = \frac{\rho}{2} (V_s^2 - V_c^2)$$

$$\begin{aligned} p_c - p_s &= \frac{0.075 \text{ lbm/ft}^3}{2 * 32.2 \text{ lbm/slug}} (78.54^2 - 100^2) \\ &= -4.46 \text{ lbf/ft}^2 \end{aligned}$$

Ex (5.49)

- **Given:** $\rho = 0.0644 \text{ lbm/ft}^3$ $V_1 = 100 \text{ ft/s}$,
and $A_2/A_1 = 0.5$, $\gamma_m = 120 \text{ lbf/ft}^3$
- **Find:** Δh
- **Solution:** Continuity equation

$$V_1 A_1 = V_2 A_2$$

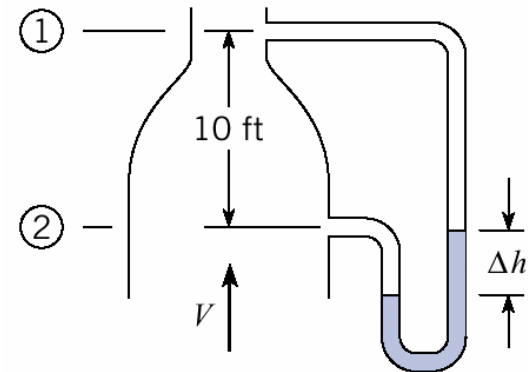
$$V_2 = V_1 \frac{A_1}{A_2} = 100 * 2 = 200 \text{ ft/s}$$

- Bernoulli equation

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

$$p_1 - p_2 = \frac{\rho}{2} (V_2^2 - V_1^2)$$

$$\begin{aligned} p_1 - p_2 &= \frac{0.0644 \text{ lbm/ft}^3}{2 * 32.2 \text{ lbm/slug}} (200^2 - 100^2) \\ &= 30 \text{ lbf/ft}^2 \end{aligned}$$



Heating ($\sim 170^\circ\text{F}$)

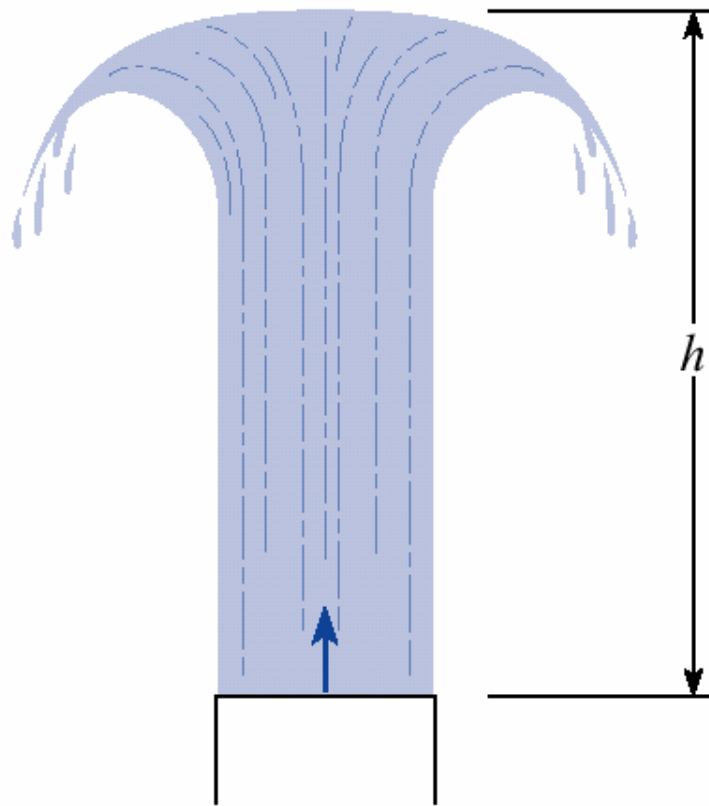
- Manometer equation

$$p_1 - p_2 = \Delta h (\gamma_m - \gamma_{air})$$

$$30 = \Delta h \frac{(120 - 0.0644) \text{ lbm/ft}^3}{32.2 \text{ lbm/slug}} * 32.2 \text{ ft/s}^2$$

$$\Delta h = 0.25 \text{ ft}$$

HW (5.51)



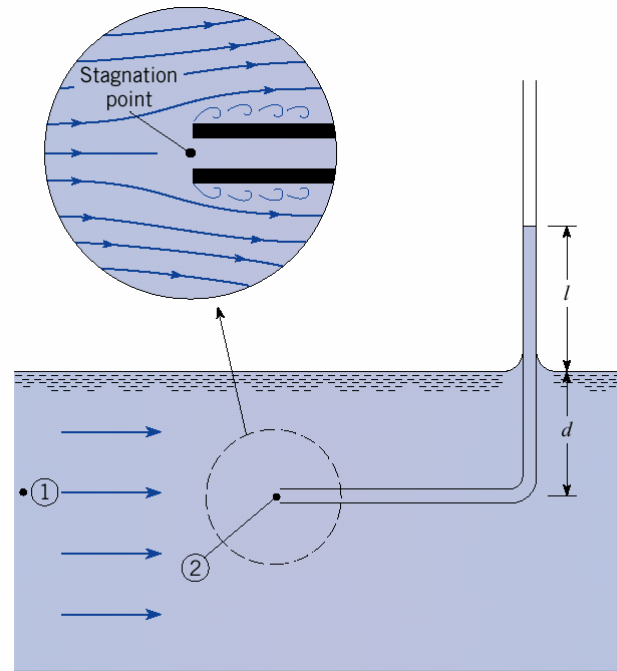
Stagnation Tube

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

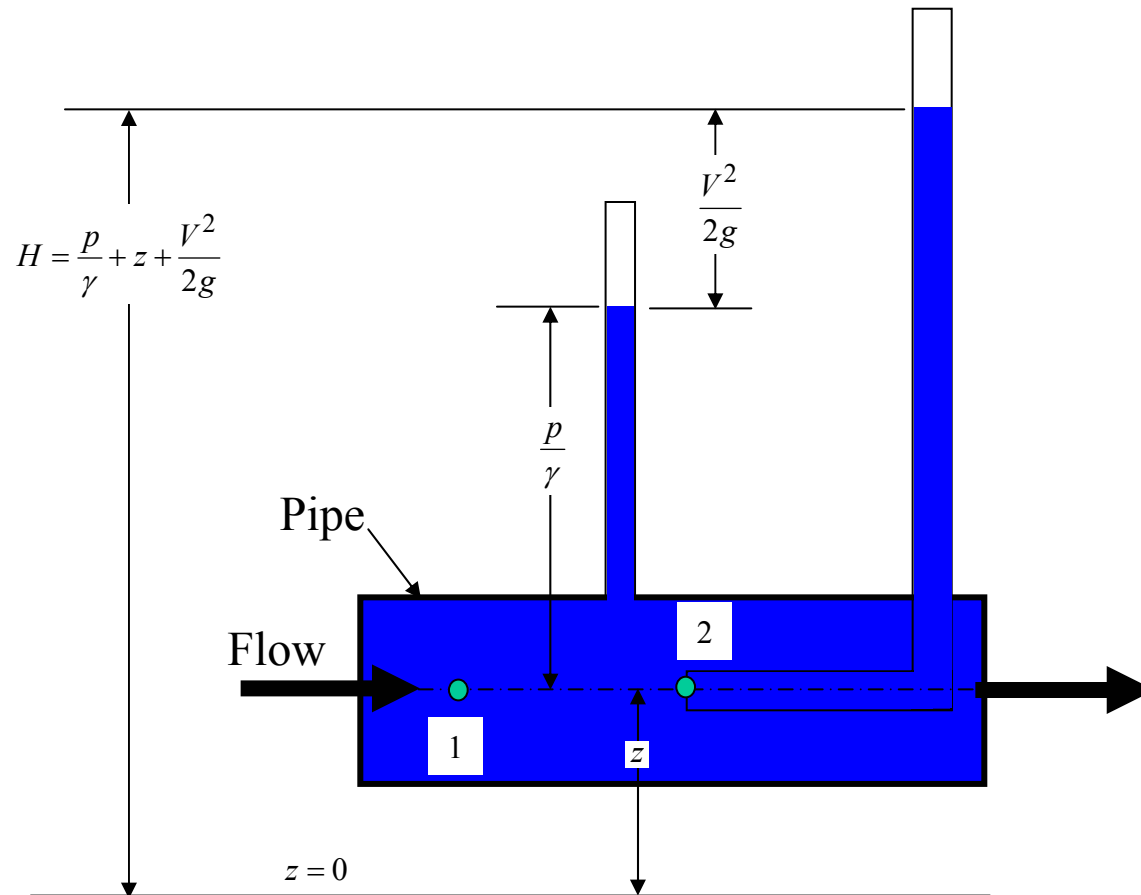
$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} = \frac{p_2}{\gamma}$$

$$\begin{aligned} V_1^2 &= \frac{2}{\rho}(p_2 - p_1) \\ &= \frac{2}{\rho}(\gamma(l + d) - \gamma d) \end{aligned}$$

$$V_1 = \sqrt{2gl}$$



Stagnation Tube in a Pipe



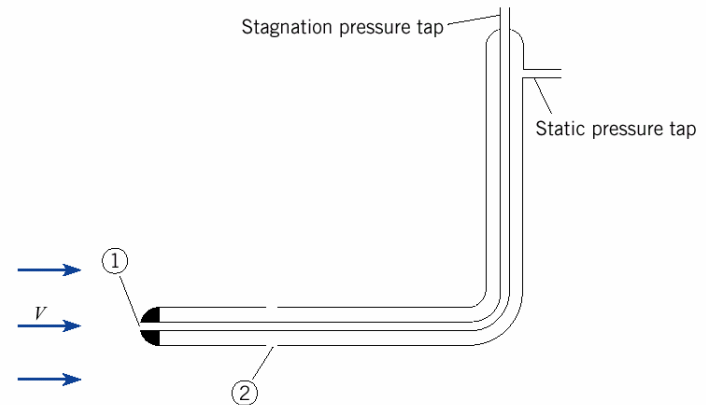
Pitot Tube

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

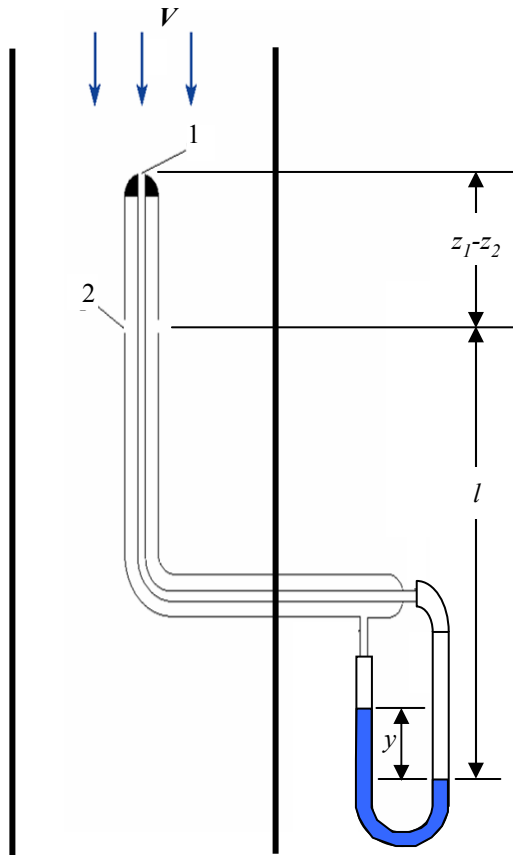
$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + \frac{V_2^2}{2g}$$

$$V_2 = \sqrt{2g\left[\left(\frac{p_1}{\gamma} + z_1\right) - \left(\frac{p_2}{\gamma} + z_1\right)\right]}$$

$$V = \sqrt{2g(h_1 - h_2)}$$



Pitot Tube Application (p.170)



$$p_1 + (z_1 - z_2)\gamma_k + l\gamma_k - y\gamma_{Hg} - (l - y)\gamma_k = p_2$$

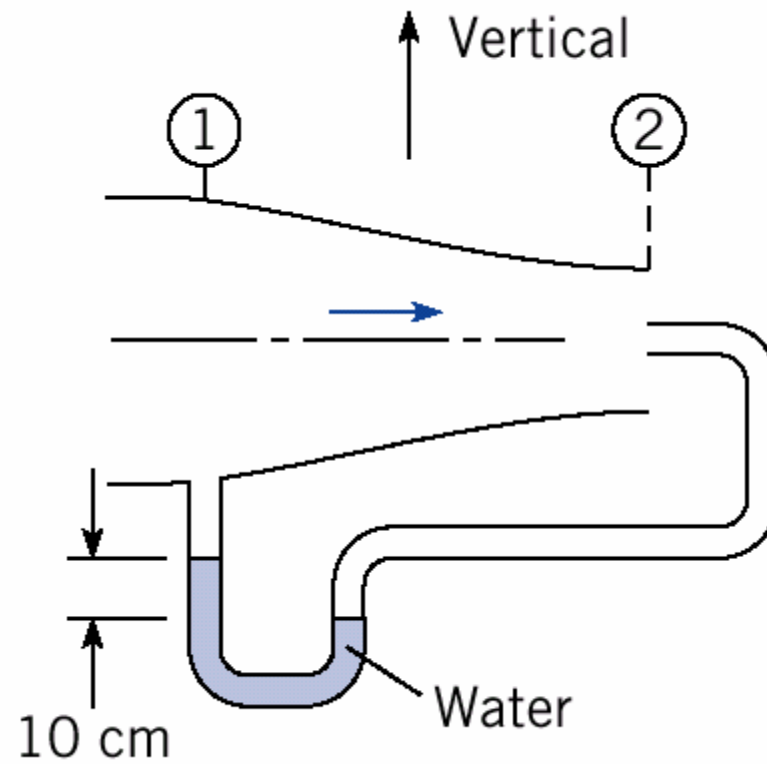
$$p_1 - p_2 = y(\gamma_{Hg} - \gamma_k) + (z_1 - z_2)\gamma_k$$

$$\frac{p_1 - p_2}{\gamma_k} + z_1 - z_2 = \frac{y(\gamma_{Hg} - \gamma_k)}{\gamma_k}$$

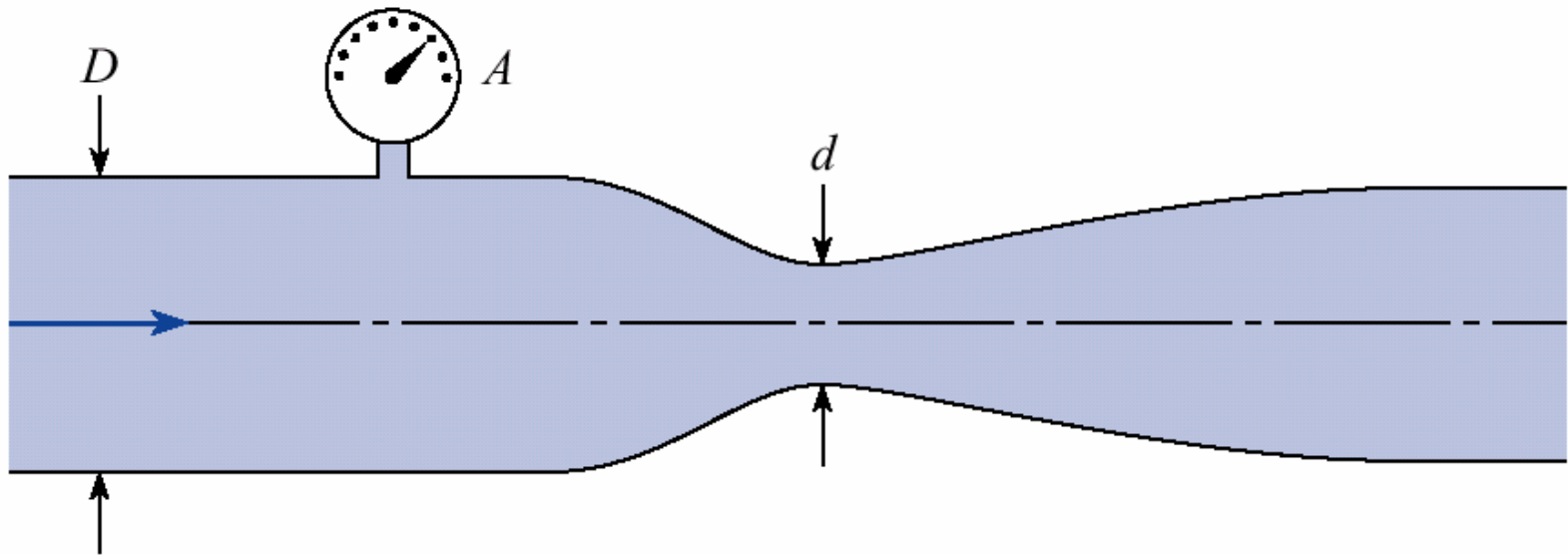
$$h_1 - h_2 = y(\gamma_{Hg} / \gamma_k - 1)$$

$$V = \sqrt{2gy(\gamma_{Hg} / \gamma_k - 1)} = 24.3 \text{ ft/s}$$

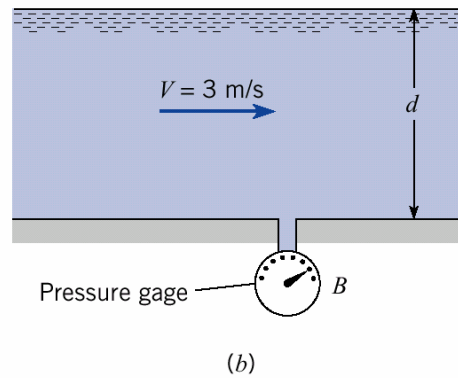
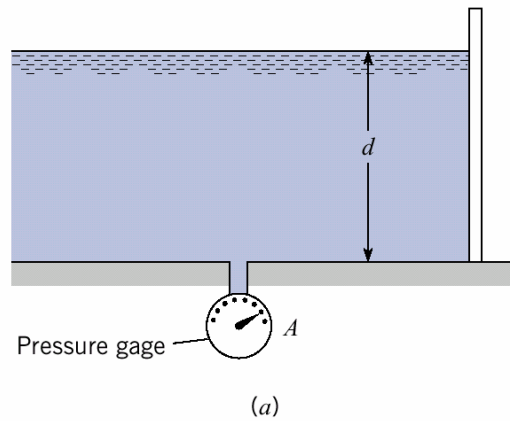
HW (5.69)



HW (5.75)



HW (5.84)



HW (5.93)

